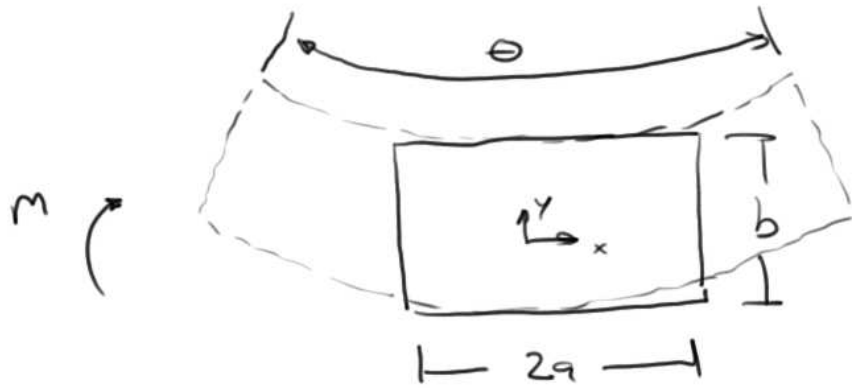


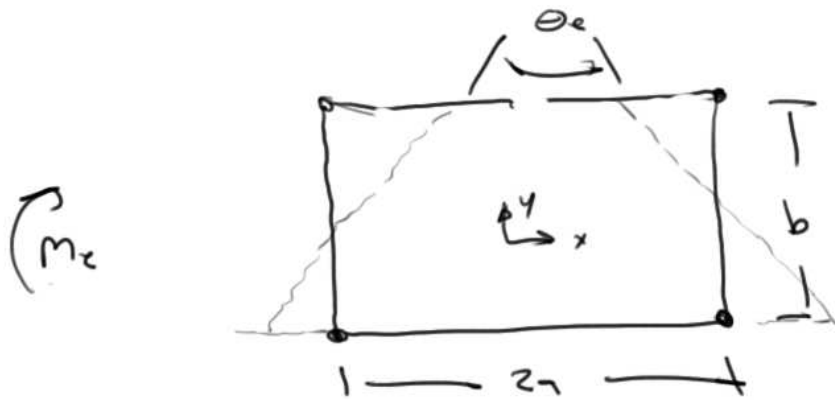
$$\text{connect} = \begin{bmatrix} 1, 2, 4 \\ 4, 3, 1 \end{bmatrix}$$

$$\begin{aligned} \text{dof} \quad 1 &\rightarrow 1, 2 \\ \quad 2 &\rightarrow 3, 4 \\ \quad 3 &\rightarrow 5, 6 \\ \quad 4 &\rightarrow 7, 8 \end{aligned}$$



$M$  Euler beam

$$\epsilon_{xx} = -\frac{\theta y}{2a}, \quad \epsilon_{yy} = \frac{\theta y}{2a}, \quad \epsilon_{xy} = 0$$



$M_e$

$$\epsilon_{xx} = -\frac{\theta_e y}{2a}, \quad \epsilon_{yy} = 0, \quad \epsilon_{xy} = -\frac{\theta_e x}{2a} \quad \& \text{ parasitic shear}$$

Elastic strain energy

$$U = \frac{1}{2} \int [\epsilon]^T C [\epsilon] dV \quad \text{where } [\epsilon] = [\epsilon_{xy} \quad \epsilon_{yy} \quad \epsilon_{xy}]^T$$

$$U = \frac{M\theta}{2} \quad U_c = \frac{M_e \theta_e}{2}$$

$M = M_e$  under plane stress  $[C] \rightarrow$  plane stress

$$\frac{\theta_e}{\theta} = \frac{1-\nu^2}{1 + \frac{1-\nu}{2} \left(\frac{a}{b}\right)^2}$$

where the term  $\left(\frac{a}{b}\right)^2$  is present due to parasitic shear

$$\lim_{\frac{a}{b} \rightarrow \infty} \frac{1-\nu^2}{1 + \frac{1-\nu}{2} \left(\frac{a}{b}\right)^2} = 0 \quad \frac{\theta_e}{\theta} = 0 \quad \text{as } \frac{a}{b} \rightarrow \infty$$

the mesh "locks"

Consider volumetric strain

$$\frac{\Delta V}{V} = \epsilon_{xx} + \epsilon_{yy}$$

Under plain strain conditions, the pressure

$$p = \frac{E_y \theta (y - 0)}{2\alpha(1+\nu)(2\nu-1)}$$

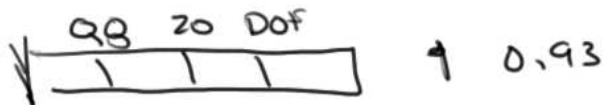
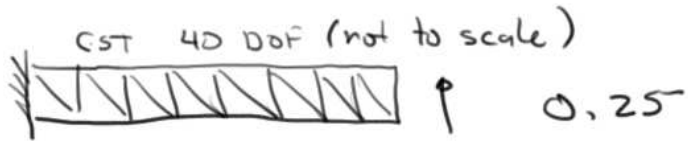
$\nu \rightarrow \frac{1}{2}$        $p \rightarrow \infty$       volumetric locking

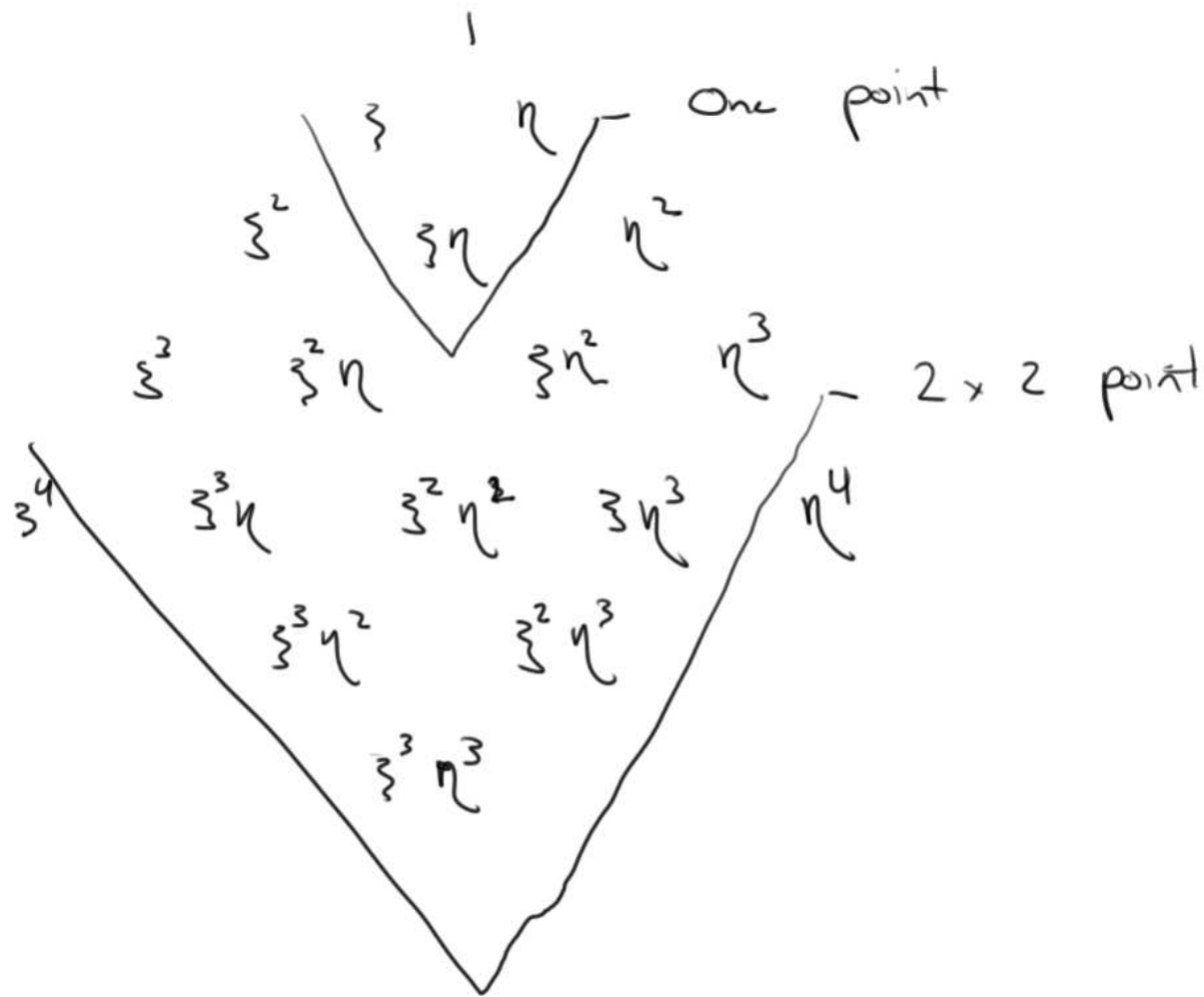
$$p_e = \frac{E_y \gamma \theta_c}{2\alpha(-1+\nu+2\nu^2)}$$

$p_e \rightarrow \infty$  , as  $\nu \rightarrow \frac{1}{2}$

"locking"  $\neq$  immovability  
= excessive stiffness

For example (Q8)





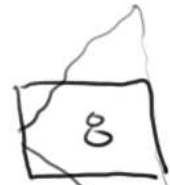
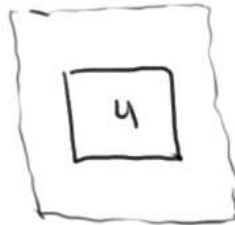
Define "full integration" = quadrature rule of sufficient accuracy to exactly integrate all coefficients of  $k^e$  of the undistorted element

$$K = \int_{\Omega^e} B^T [C] B J_3 h e d\Omega \Rightarrow \mathcal{O}(\xi^2 \eta^2) \text{ for Q4 } \quad 2 \times 2 \text{ rule}$$

## Under integration

AKA "reduced integration"

- Use a rule that is less than exact
- Reduces computation time
- Can offset parasitic shear
- Introduce a defect i.e. spurious modes, singular modes, zero-energy modes, hourglass modes.



Recall:  $\epsilon_{xy} = -\frac{\theta_c x}{2a}$  in bending